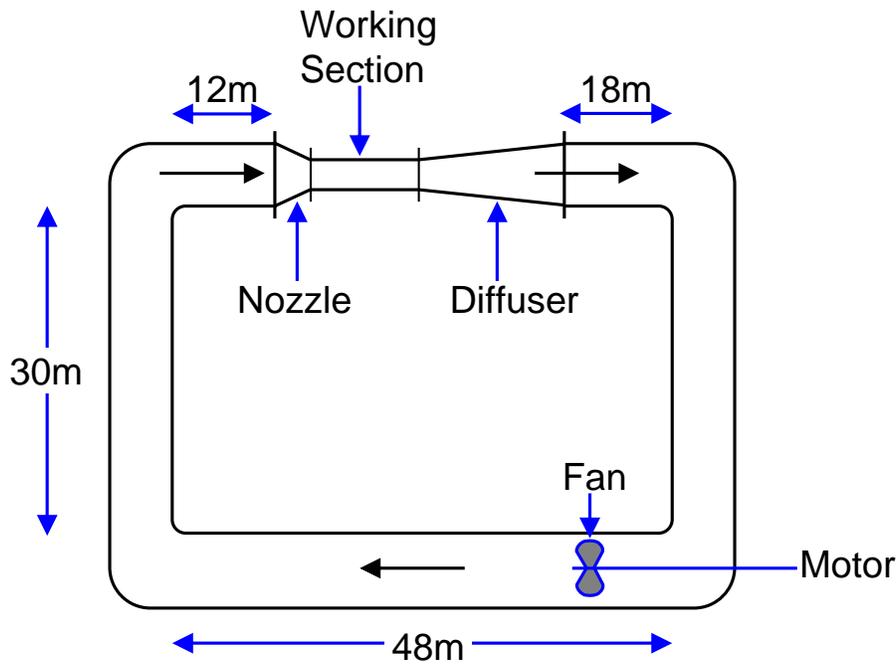


Solution to Problem 205D

The total pressure losses in the circuit are composed of several contributions:

- Total pressure loss due to length of pipe, Δp_{el}
- Total pressure loss due to the bends in the pipe, Δp_b
- Total pressure loss in nozzle, working section and diffuser, Δp_n

The power output of the fan must be increased to offset the losses Δp with power increase P .



The total pressure loss due to the length of the pipe will be due to the following total length of 6m pipe:

$$L_{el}^* = 48 + 30 + 30 + 12 + 18 = 138 \text{ m}$$

The total pressure loss in the bends of the pipe can be taken into account by the addition of the following effective length of pipe:

$$L_b^* = 4 \text{ bends} \cdot 20D_{pipe} = 80D_{pipe} = 480 \text{ m}$$

Consequently the total pressure loss due to the total effective length of the 6m pipe is:

$$\Delta p_{L^*} = \Delta p_b + \Delta p_{el} = \frac{fL^*}{D} \frac{1}{2} \rho U^2 = 1.03 \rho U^2$$

with

$$L^* = L_{el}^* + L_b^* = 618 \text{ m}$$

The total pressure loss in the nozzle, working section and diffuser is computed from:

$$\Delta p_n = \frac{1}{5} \frac{1}{2} \rho u^2$$

where the velocity in working section $u = 80 \text{ m/s}$. The volume flow rate, Q , is given by :

$$Q = uA_{\text{workingsection}} = UA_{\text{pipe}} = 180\pi \text{ m}^3/\text{s}$$

so that the velocity in the 6m pipe U is:

$$U = \frac{D_{\text{workingsection}}^2}{D_{\text{pipe}}^2} u = \frac{u}{4}.$$

Then the total pressure loss in the nozzle, working section and diffuser is:

$$\Delta p_n = \frac{8}{5}\rho U^2 = 1.6\rho U^2$$

and therefore the total total pressure loss in the circuit, Δp , is

$$\Delta p = (1.03 + 1.6)\rho U^2 = 2.63\rho U^2$$

This must also be the total pressure rise across the fan.

The total power input P to the fan is then given by:

$$P = \frac{Q\Delta p}{\eta} = 8.923 \times 10^5 \text{ kg m}^2/\text{s}^2$$

with flow rate Q , total pressure loss Δp and given efficiency $\eta = 0.8$. Thus

$$P = 1.196 \times 10^3 \text{ HP}$$